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The design and commissioning of the first, circular, combined current and wave test basin.

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Abstract—In 2010 Bryden, Ingram and Wallace applied to the UK Engineering and Physical Sciences Research Council (EPSRC) for funding to construct the all waters combined current and wave test facility. Funding was awarded and over the next three years the FloWave TT facility was constructed. At its heart is a 25m diameter, circular basin, equipped with 168 force feedback wave makers, 28 bidirectional impellers and a liftable floor. The 2m deep test section is designed to generate currents (at 0.8m/s) and 700mm high, 2s period, waves from any relative directions. Allowing a model to be subjected to scale tests over the full tidal ellipse simultaneously with multi-directional waves.

Construction of the facility was completed in November 2013, and calibration is currently in progress. Initial work has shown that maximum flows of 2m s⁻¹ can be achieved across the test section, while the circular wave maker array allows very large focused waves to be created.

This paper describes the FloWave facility, its construction and commissioning and presents some preliminary results.

I. INTRODUCTION

FloWave (Figure 1) is the first basin in the world to allow the combination of waves and currents in any relative direction. It has been designed to provide a proving ground and development facility for ocean energy devices (including fixed and floating offshore wind turbines, and wave and tidal energy converters). The basin has been sized to allow small feedback wave makers, 28 bidirectional impellers and a liftable floor. The 2m deep test section is designed to generate currents (at 0.8m/s) and 700mm high, 2s period, waves from any relative directions. Allowing a model to be subjected to scale tests over the full tidal ellipse simultaneously with multi-directional waves.

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II. FLOWAVE CONSTRUCTION

A. Design

The design of the facility was inspired by a combined current and wave test basin design described by Salter [1] (Figure 2) but never constructed. Salter’s basin utilised a giant Voith-Schneider impeller with undershot wave makers, in contrast, the FloWave basin uses 28 independently controlled 1.7m diameter impellers. Rather than under shooting the wave makers flow is introduced in front of the wave makers using shaped transition vanes to generate a region of quiescent water in front of the wave makers bounded by a strong turbulent shear layer [2]. The basin is designed to operate at scales of between 1:20 and 1:40. This region is chosen because the performance (with respect to Reynolds number) of tidal rotors under such conditions are comparable (in terms of tip speed ratio and coefficients of power) to full scale rotors (see Figure 3).

The most energetic sites for tidal energy extraction in the UK (in the Orkney Islands, the Pentland Narrows and the Sound of Islay) are characterised by water depths of between 20m and 70m with current velocities in excess of 5m s⁻¹. Richmond et al report that the turbulence intensities for flows in tidal channels are around 15% [3] . Wave devices are typically deployed in water depths of between 10m and 100m with wave heights of up to 30m. Large atlantic swell waves at such sites can have wave periods in excess of 20s. Applying Froude scaling, we determine that the basin should:

- achieve flow speeds of 0.8 m s⁻¹ with an associated turbulence intensity of circa 7%,
- provide waves of 700mm with periods of 2s, and
- have a water depth of between 2 – 2.5m.

The decision to require low turbulence intensity levels of 7% is to allow higher levels of turbulence to be provided in experiments as “designer” turbulence. This is done by placing structures in front of the tidal rotors to create large scale, or high intensity, eddy structures, much in the same way as is done in wind tunnels.
1) Flow conditioning: The use of a strong shear layer to create a quiescent region in front of the wave makers is critical to the successful operation of the facility. The wave makers deployed in the FloWave basin use force-feedback [4], [5] absorb incoming waves and the presence of flowing water in front of the wave makers will modify the force, causing wave absorption to fail. It is therefore critical that the wave makers are located in quiescent water. One of the main deficiencies of the under-shot “race-track” design proposed by Salter [1] is that the wave makers are not isolated from the current. Robinson et al [6], [2] suggested the use of a strong turbulent shear layer in front of the wave maker to create a quiescent region. This proposal is based on a comprehensive series of computational fluid dynamics (CFD) simulations and experiments conducted both in the curved wave basin at Edinburgh University and in a specially constructed wave/current flume. Figure 4 shows a single frame from a transient CFD calculation performed using Flow3D to illustrate the concept. The wave maker is isolated from the flowing water by a strong-shear layer, beyond which the transformation of the waves, due to Doppler shifting, is clearly visible. The CFD simulations have been validated using PIV and ADV measurements and the shear-layer method is shown to work at both the inflow- and outflow- ends of wave-current flume (Full details are given in [2]).

2) Plane waves in circular tanks: The, so-called, snake theory which allows the generation of plane waves at an angle to a segmented wave maker [7], [8] is simply extended to circular wave tanks and has been used in both the Japanese AMOEBA tank [9], [10] and the NMRI deep sea basin in Tokyo Japan. Whilst the AMOEBA basin is very small (D=1.6m), the NMRI basin has a diameter of 15m, just over half that of FloWave.
Figure 7 shows the installation of the reinforcement for the step on which the wave makers are mounted. This step, located 2m below the still water level, marks the transition between the 25m diameter and 30m diameter sections of the tank. The wider sections is necessary to accommodate the maintenance and access gallery behind the dry-backed, flap-type, wave makers.

The civil construction began in late 2011 and was sufficiently complete by January 2013 that the E&M installation could begin. Handover of the building from the civil contractors took place in November 2013.

C. E&M Installation

The design and installation of the electrical and mechanical equipment for the 168 wave makers, 28 flow drives and the associated control was overseen by Edinburgh Designs Ltd (www.edesign.co.uk). EDL is a spin out company from the University of Edinburgh and have been specialists in the design, manufacture and installation of a wide range of wave and tidal generators since 1987. Installation began with the erection of the wave making flaps, followed by the 90° turning vanes (Figure 8). Each of the 28 flow drive units has its own set of turning vanes and baffles are installed between them to prevent short-circuiting of the flow path. Once the turning vanes had been erected the flow drive units were positioned (Figure 9) and then the floor was installed (Figure 10). The between the lower and upper skins the floor frame holds a large number of floatation tanks, sufficient to provide enough buoyancy to ensure the floor can support a load of 5 tonnes without significant movement, the cables which hold the floor down against the buoyancy force have been sized so this can be applied as a point load on the edge of the floor.

In addition to the mechanical equipment described above, control cables and power had to be installed for the 168 wave makers and 28 flow drives. The wave makers draw a maximum of 300kW and the flow drives a maximum of 700kW, allowing for a diversity factor, making the total electrical demand 1MW. It is interesting to note that when the wave makers are in “absorbing” mode the energy is injected back into the electricity supply, this means that if the tank is instructed to stop making waves for a few minutes we have a grid connected 300kW wave power machine!

Once the E&M Installation was complete the tank was filled with 2.5Ml of potable water and commissioning tests began. Figure 11 shows the floor in the raised position.

III. COMMISSIONING

A. Current system

Simulations by Robinson et al [2] show that by individually controlling each of the 28 flow drive units, linear flow can be achieved across the test section of the basin at the desired velocity (Figure 12).

Once the flow drive system had been installed and the tank filled a series of measurements were conducted using a single-axis electromagnetic flow meter to check this and to derive a calibration curve relating motor speed (in RPM) to flow speed (in ms⁻¹). Measurements were made using a Valeport Model 801 single-axis electromagnetic open channel flow meter.
The Model 801 is a high precision instrument which gives accurate readings ($\pm 0.5\% + 0.005\text{ms}^{-1}$) over a flows with velocities of $\pm 5\text{ms}^{-1}$. In the calibration tests the “flat-type” probe was used which has a cylindrical sensing volume $20\text{mm} \times 10\text{mm}$. It has a sampling frequency of $1\text{Hz}$ and can average data over a period of between 1 and 60 seconds, in accordance with the manufacturers recommendations sampling periods $30\text{s}$ were used to ensure stability of the measurements in a turbulent flow environment. The Model 801 also records the sample’s standard deviation.

For these, current only, tests the wave makers were inactive and parked against their backstops. Thus the water level for these tests is approximately $90\text{mm}$ lower than the nominal $2\text{m}$ working depth. The probe was placed at a depth of $0.5\text{m}$ below the still water level (i.e. approximately $1.4\text{m}$ from the tank floor) and aligned with the flow direction. Velocity measurements were taken from the centre of the tank out to a radius of $9.5\text{m}$ in increments of $0.5\text{m}$, by moving the gantry with the flow meter fixed in place. Throughout the tests the maximum flow drive speed was set to $100\ \text{rpm}$ which corresponds to a flow velocity of $1\text{ms}^{-1}$ which is very close...
to the velocity of 0.8\text{ms}^{-1} used to design the flow transition vanes.

Figure 13 shows the measured flow velocity profile across the basin together with the 95% confidence intervals, estimated as \( \bar{v} \pm 1.96\sigma \), plotted against radial distance from the centre of the basin in the crossflow direction. The plot shows good agreement with the profile predicted using CFD (Figure 12). The results also show that uniform flow is achieved out to \( r = 5.5\text{m} \), demonstrating that the central 11m of the tank have approximately uniform flow conditions.

Initial calibrations based on the central velocity (at \( r = 0 \)) show that the flow velocity varies linearly with the maximum flow drive rpm up to the maximum speed of 200 rpm, which equates to a flow velocity of 2.0\text{ms}^{-1}. The final maximum flow velocity for the basin is expected to be around 1.6\text{ms}^{-1} as additional flow conditioning membranes need to be installed underneath the flow transition vanes to further reduce the turbulence intensity (which is currently around 10%).

B. Wave system

Preliminary calibration of the wave system is done by generating long crested regular waves over a number of directions. Once the generation and absorption characteristics of the tank have been established. Figure 14 shows photographs taken from the centre of the truss on the north side of the tank during tests with long crested regular waves. In both cases the wave amplitude is the same (circa 400mm) but the directions of wave propagation are at 0\(^\circ\) and 90\(^\circ\) with respect to the E-W axis of the tank (we define 0\(^\circ\) as being away from the control room window).

Figure 15 shows the a JONSWAP wave spectrum (\( T_p = 2s, H_{m0} \approx 300\text{mm}, \gamma = 3.3 \)) with a uniform propagation direction of 270\(^\circ\) and a 30\(^\circ\) cosine spreading function. Random waves are generated using an inverse Fourier transform method with randomised component phases and so are completely deterministic. Currently these tests are being repeated with a wave gauge array deployed from the gantry to calibrate the tank transfer function and to check the directionality of the waves.

A final check on the quality of wave making in circular tanks can be performed by focussing concentric circular waves onto the centre of the basin. The idea is to generate a circular, breaking wave, in the centre of the tank which creates a circular jet. The jet is formed by creating waves of decreasing frequency at the wave-makers in such a way as they all arrive at the centre of the tank at the same instant. The process can be thought of as the “inverse stone throwing problem” where instead of watching the ripples spread out from the point of impact, we simply create the ripples at the edge of the tank and run time backwards!

The formation of the up rushing jet is a highly non-linear phenomena which is thought to be generated by a circular
version of the “flip-through” process described by Peregrine and Cooker (see [11]). Flip-through was described in relation to the impact of breaking waves on vertical walls and has been shown to cause extreme up rushing velocities, many times the inshore wave celerity. Indeed, experiments by Bruce et al [12], conducted on a vertical wall in the large wave flume in Barcelona, measured velocities in excess of 20 times the inshore wave celerity. In terms of testing the wave generation in the FloWave basin, the created jet should be a completely circular column of water show no obvious asymmetry.

Figure 16 shows the the “phase-locking” of the wave components approximately 2s before impact and the rapidly up-rushing jet just after impact. In these tests a maximum current of 1.2A was supplied to the wave maker motors (which are rated for 12A), and the resulting jet hits the ceiling of the tank hall (10m above the water surface). Provided the surface of the basin is quiescent prior to the start of the experiment, both the up rushing column of water and the incident wave crests show almost perfect circularity and there are no defects, or scars, observed in the uprushing column of water until the jet starts to fragment. This test shows the wave making process is well controlled and the waves of a very high quality.

C. Initial WEC deployment

As part of the calibration procedure for the FloWave basin two Scottish developers (one wave energy company and one tidal energy company) have been given the opportunity to conduct preliminary tests of their device in the basin. The wave company is AlbaTERN (www.albatern.co.uk). AlbaTERN’s WaveNET wave energy converter is a modular array of their SQUID units, which can also function as a stand-alone wave energy device. SQUID devices are currently being developed at 7.5kW nameplate capacities. The SQUID device is road transportable and can be launched from a wide variety of locations. AlbaTERN are currently developing a 75kW SQUID unit. An WaveNET array of 10 SQUID units was deployed in the basin and tested under a range of long-crested regular and irregular wave conditions (Figure 17).

The primary purpose of the test was to demonstrate the installation process and to check the mooring arrangement used. Following these tests AlbaTERN are designing the mooring system to include force transducers on the moorings and to instrument the SQUID energy conversion units.

IV. Conclusion

The FloWave basin, constructed at the University of Edinburgh, is the worlds first completely multidirectional, combined current and wave basin. The 25m diameter basin is able to create, highly repeatable waves using 168 individually controlled force feedback wave makers. The wave makers are designed for waves with an amplitude of 700mm and a period of 2s. Preliminary tests on the wave making indicate that high quality, highly repeatable, waves can be generated in the basin. Further work is on-going to characterise the waves using an array of wave gauges to check both the quality and directionality of the waves.

In terms of current, calibration measurements have shown that the basin can exceed it’s design velocity of 0.8ms⁻¹ and that uniform flow can be created across a circa 11m diameter section in the centre of the basin. Calibration tests have also shown that the flow speed varies linearly with the motor speed, providing a high degree of control over the current velocity. Further measurements are presently being undertaken to characterise the boundary layer across the floor of the
basin. The current measurements demonstrate that the CFD simulations and flume experiments used to design the turning vanes in the basin have been successful and that a quiescent region of water is created in front of the wave makers allowing the combination of waves and currents.

Preliminary tests with a wave energy device developer have shown that the basin provides a sufficiently large area for array tests to be performed and that models can quickly and easily be installed and removed.

FloWave thus represents the first time that realistic ocean conditions, typical of those found at marine energy deployment sites, can be created, under controlled conditions, in the laboratory. Over the next few months further calibration work will be conducted on both the wave and current systems and early experiments with combined conditions will be performed.

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